

Dec. 12, 1967

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STRAND BRAKE FOR LOAD LOWERING ASSEMBLY

Filed March 10, 1966

3 Sheets-Sheet 1

FIG. 2

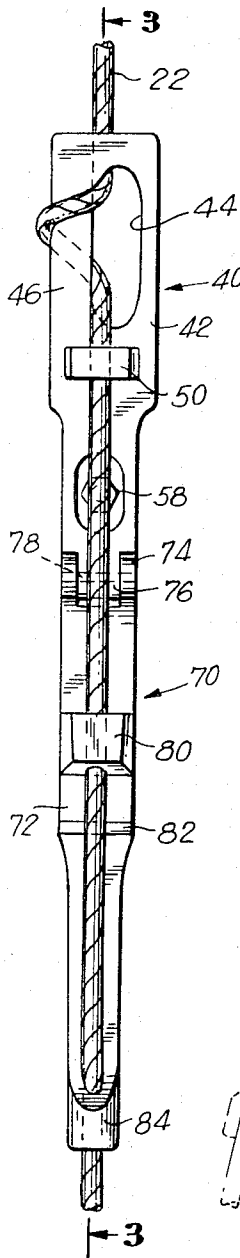


FIG. 3

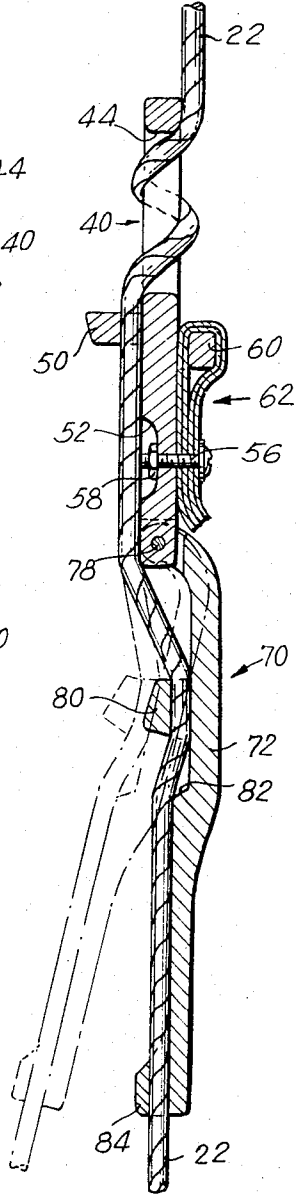


FIG. 1

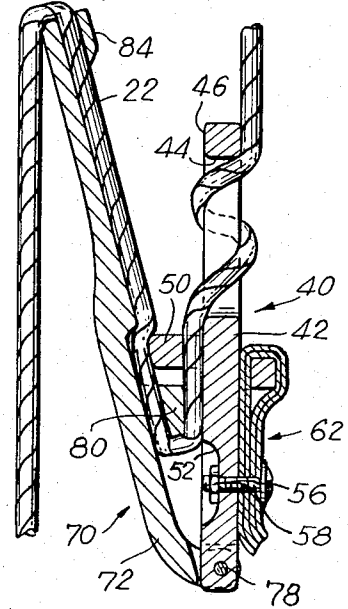
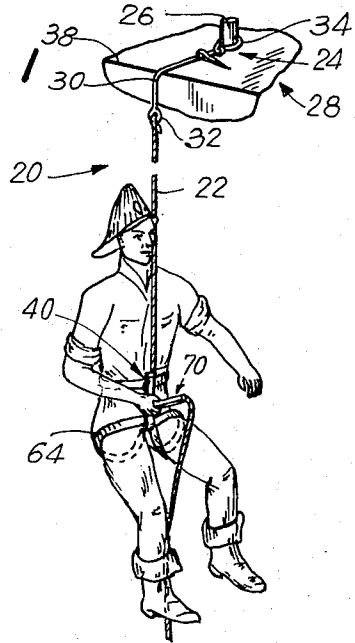


FIG. 4

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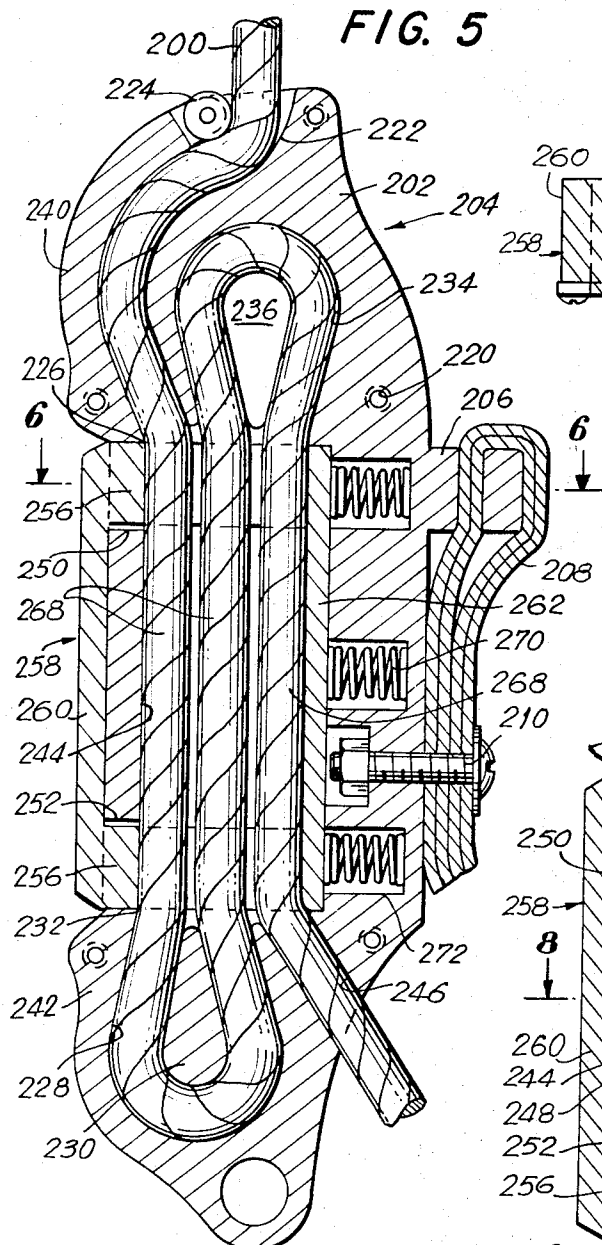


FIG. 6

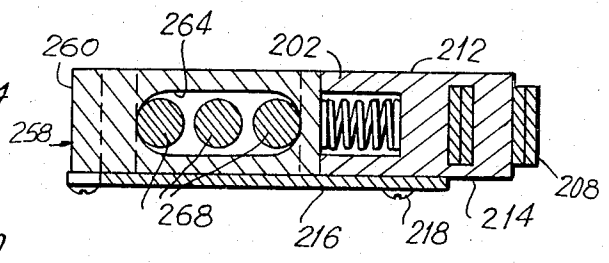


FIG. 7

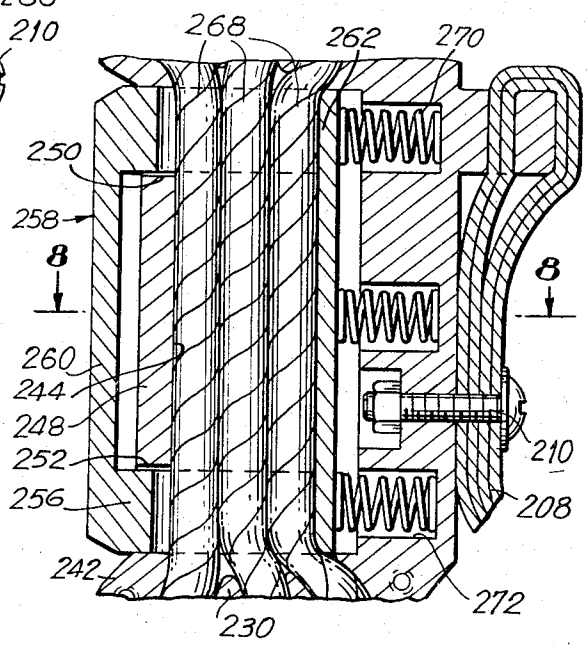
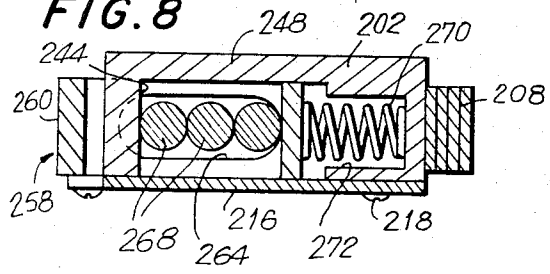


FIG. 8



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FIG. 9

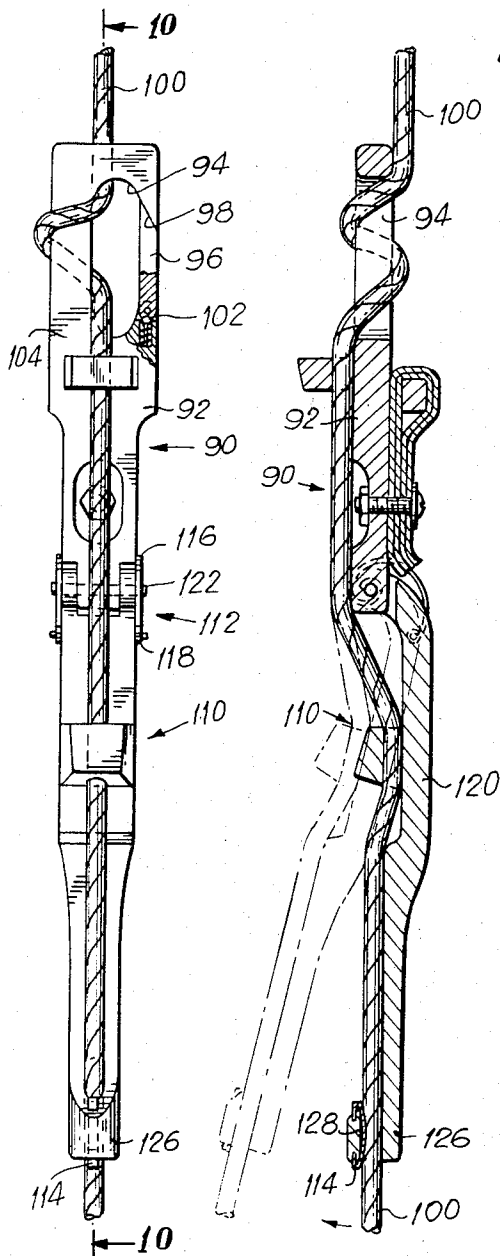


FIG. 10

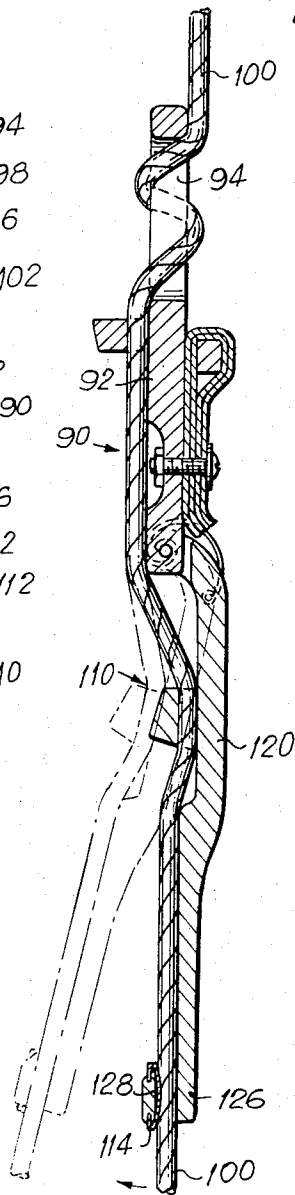
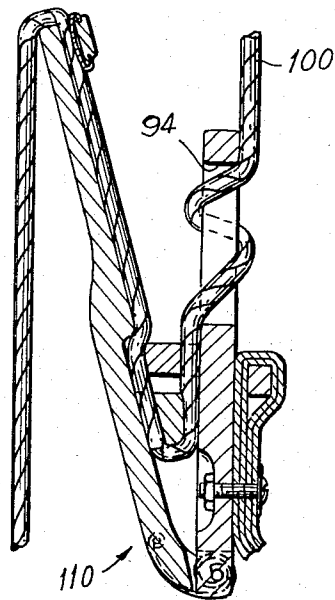


FIG. 11



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**STRAND BRAKE FOR LOAD LOWERING
 ASSEMBLY**

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 11 Claims. (Cl. 188—65.3)

ABSTRACT OF THE DISCLOSURE

A load-lowering assembly, particularly of a type which is suitable for fire-fighting purposes. The assembly includes a friction slider means for frictionally engaging and sliding down a rope and for carrying a load which urges the slider means downwardly along the rope. A manually operable brake means is operatively connected with the slider means for automatically assuming a braking position braking movement of the slider means and a load carried thereby downwardly along the rope, this brake means being manually displaceable with respect to the slider means from the braking position to a release position releasing the slider means for movement downwardly along the rope. The slider means and the brake means respectively include a pair of elongated bodies both of which are formed with openings through which the rope passes for frictionally engaging these bodies. The slider means has an upper end and the brake means is pivotally connected to the slider means below the upper end thereof. When the brake means is in its release position it extends downwardly from its pivotal connection to the slider means, and the brake means, when released by the operator, coacts with the rope for responding automatically to tension in the rope to be pulled upwardly by the tension in the rope to its braking position, this brake means when in its braking position extending upwardly from the above pivotal connection to the slider means and being situated alongside of the body of the slider means.

The present invention relates to load-lowering assemblies.

In particular, the present invention relates to that type of assembly in which an individual can lower himself downwardly along a rope.

Although assemblies of this general type are known, in mountain climbing, for example, it is essential with the conventional assemblies of this type that the operator manually engage the structure for stopping the lowering of the load at a selected elevation along the rope. Thus, it is essential that an individual who is being lowered downwardly along such a rope have at least one hand free in order to manipulate the structure for terminating the lowering at a selected elevation. A further difficulty encountered with the conventional structure of the above type is that it is not always possible to regulate the speed with which the load is lowered. Since the individual who is being lowered will release the structure to permit it to move downwardly along the rope, there is practically no control on the speed of movement downwardly along the rope, and the individual who is being lowered only manipulates the structure in order to stop the downward movement.

Structure of the above general type not only has the above drawbacks but in addition it includes the drawback of preventing an individual who is being lowered from having both hands free when his movement down the rope is terminated. Thus, if an individual moving downwardly along the rope should wish to perform certain operations part of the way down, only one hand would be available for these operations since the other hand would neces-

sarily be required to control the structure which prevents movement downwardly along the rope.

It is, therefore, a primary object of the present invention to provide a load-lowering assembly which will avoid the above drawbacks.

In particular, it is an object of the invention to provide a load-lowering assembly which enables the individual who is being lowered to stop his downward movement at any selected elevation while at the same time having both hands free at wherever elevation the lowering is terminated.

An additional object of the present invention is to provide an assembly of this type which is particularly suitable for fire-fighting purposes.

It is especially an object of the present invention to provide a structure of this type which is eminently suitable for rescuing operations in connection with fires or other calamities.

Furthermore, it is an object of the invention to provide a structure which is exceedingly simple and inexpensive while at the same time very reliable in operation and extremely convenient to use.

Primarily, with the structure of the invention the rope down which the load is to be lowered is in frictional engagement with a friction slider means which slides downwardly along and frictionally engages this rope. A brake means is also in frictional engagement with the rope, and in accordance with the present invention the brake means will automatically assume a braking position preventing movement of the slider means downwardly along the rope. This brake means is manually operable so that it can be displaced by the operator from its braking position to a release position where the brake means releases the slider means for movement downwardly along the rope, and in this way whenever the brake means is released it will automatically terminate the movement of the slider means downwardly along the rope, so that the operator will have both hands free while situated at a selected elevation along the rope.

The invention is illustrated by way of example in the accompanying drawings which form part of the application and in which:

FIG. 1 is a fragmentary schematic illustration of the manner in which the structure of the invention can be used;

FIG. 2 is a front elevation of one possible embodiment of an assembly according to the invention;

FIG. 3 is a longitudinal sectional elevation of the structure of FIG. 2 taken along line 3—3 of FIG. 2 in the direction of the arrows;

FIG. 4 shows the structure of FIG. 3 in its braking position;

FIG. 5 is a longitudinal sectional elevation of another embodiment of a structure according to the present invention;

FIG. 6 is a sectional plan view taken along line 6—6 of FIG. 5 in the direction of the arrows;

FIG. 7 fragmentarily illustrates part of the structure of FIG. 5 in a braking position;

FIG. 8 is a sectional plan view taken along line 8—8 of FIG. 7 in the direction of the arrows;

FIG. 9 is a front elevation of another embodiment of a structure of the type shown in FIG. 2;

FIG. 10 is a longitudinal sectional elevation of the structure of FIG. 9 taken along line 10—10 of FIG. 9 in the direction of the arrows; and

FIG. 11 shows a structure of FIG. 10 in a braking position.

Referring now to FIG. 1, the load-lowering assembly 20 illustrated therein includes an elongated rope 22 which carries at its upper end a fixing means 24 by which it

can be fixed to a selected support. In the example shown in FIG. 1 the fixing means 24 fixes the upper end of the rope 22 to a pipe 26, for example, situated at the top of a building 28. The fixing means 24 is in the form of a wire cable 30 having at its lower end an eye 32 through which the top end of the rope 22 passes and to which the top end of the rope 22 is fixed, and this cable 30 carries at its free end a ring 34 so that the cable can pass through this ring to form a loop extending around the pipe 26. However, the cable 30 can also carry a spike capable of being driven into any suitable beam of wood or the like for fixing the top end of the rope 22. It is preferred to use a wire cable 30 because the top end of a building 28 will generally have a sharp-edge 38 forming the outer upper periphery of a concrete ledge or the like over which the cable 30 can pass. Since the cable is in the form of a twisted wire it will not easily become frayed, while a conventional rope would become frayed by the edge 38, thus rendering the support of the load more precarious than by the use of a wire cable 30.

In the example shown in FIG. 1, the assembly 20 is used in connection with fire-fighting operations, and there is indicated in FIG. 1 a fireman lowering himself downwardly along the rope 22 so that, for example, when the fireman is situated at the elevation of a window of a building where an individual is located to be rescued, the fireman can carry out these rescue operations in a highly efficient manner with the structure of the invention.

Referring now to FIGS. 2-4, the structure of the invention includes an elongated friction slider means 40 in the form of an elongated metal body 42 formed in any suitable way as by a casting made of iron, for example, in a suitable mold. The elongated friction slider body 42 has at its upper end an opening 44 through which the rope 22 is threaded in the manner shown in FIGS. 2 and 3. Thus, the rope 22 is passed through the opening 44 and around the lateral portion 46 of body 42. The lateral portion 46 is wider than the opposed lateral portion, so that it is better capable of cooperating with the rope 22.

The elongated body 42 which constitutes the friction slider means 40 integrally carries a projection 50 formed also with an opening through which the rope 22 passes so as to frictionally engage the body 42. Below the projection 50 the body 42 is formed with a recess 52 which receives the end of a screw 56 together with a nut 58 on this screw.

An attaching means 60 is provided for attaching a harness 62 to the friction slider means 40, and this attaching means 60 is in the form of a metal loop integral with the body 42 and through which bands of the harness 62 pass. The harness 62 is conventional and includes, for example, a pair of leg loops 64 (FIG. 1) adapted to pass around the legs of the operator in the manner indicated in FIG. 1.

In the position of the parts shown in FIGS. 2 and 3, a brake means 70 is situated below and extends downwardly from the friction slider means 40. The brake means 70 will, in accordance with the invention, automatically assume a braking position shown in FIG. 4 when the operator releases the structure of the invention. Thus, the structure of the invention is shown in its release position in FIGS. 2 and 3, and it is necessary for the operator to maintain the structure of the invention in the position of FIGS. 2 and 3. The brake means 70 includes an elongated body 72 which also may be in the form of a metal casting. This body 72 is formed with a pair of spaced ears 74 which are formed with aligned bores and which receive the bottom end portion 76 of the friction slider body 42, in the manner shown most clearly in FIG. 2. The portion 76 is formed with a bore passing therethrough in alignment with the openings of the ears 74, and a pivot pin 78 extends through the bore of the portion 76 and the openings of the ears 74, so that in this way the brake means 70 is pivotally connected with the friction slider means 42 at an elevation lower than the upper end there-

of. In the release position of the brake means 70 extends downwardly from its pivotal connection 78 with the friction slider means 40, while in its braking position, the brake means 70 extends alongside of the friction slider means 40, as indicated in FIG. 4.

The elongated body 72 has an integral projection 80 formed with an opening through which the rope 22 passes, so that the rope 22 is also in frictional slidable engagement with the body 72 of the brake means 70. Just beyond the projection 80 the body 72 is formed at its face which carries the projection 80 with a shoulder 82 for a purpose described below. The rope 22 extends downwardly beyond the shoulder 82, in the position of the parts shown in FIGS. 2 and 3, and at its free end the body 72 is formed with an elongated bored portion 84 through which the rope 22 passes so as to be in frictional engagement with the body 72 at its free end portion also, guided through the bore at the end of this free end portion.

In using the above-described apparatus of FIGS. 1-4, the operator will initially fix the top end of the rope 22 to a selected support such as the pipe 26 in the manner described above by way of the fixing means 24.

Then the operator will put on the harness 62, after the rope 22 has been threaded through the friction slider means and brake means to have the position indicated in FIGS. 2-4. Of course, before the fixing means is attached to a selected support the rope has already been threaded through the apparatus of the invention so that the assembly of the invention is ready for use at any time.

With the brake means 70 maintained in the release position shown in FIGS. 2 and 3 the operator can slide freely downwardly along the rope 22 which by being passed a selected number of turns around the portion 46 of the slider 42 will provide a selected frictional resistance to downward movement, thus providing a predetermined speed of downward movement of the load when the brake means is in the fully released position thereof shown in FIGS. 2 and 3.

Assuming now that the operator wishes to terminate his downward movement when reaching a given elevation, such as the elevation of a window where an individual to be rescued is located, the operator will simply release the body 72. The result is that due to the frictional resistance to movement of the rope through the opening of the projection 80 as well as through the bore of the free end portion 84 of the brake means and through the opening of the projection 50, the slider continues to move downwardly for a slight distance along the rope 22, but the rope 22 itself pulls on the projection 80 so as to automatically turn the body 72 into the position of FIG. 4. In this position it will be noted that the projection 80 is located beneath the projection 50 pressing the rope against the friction slider body 42, while at the same time the projection 50 presses the rope 22 against the brake body 72 and in particular against the portion of the brake body 72 which is provided with the shoulder 82 which becomes situated just above the projection 50 in the braking position shown in FIG. 4. As a result there is a very high frictional engagement with the rope preventing further downward movement of the structure which will thus remain automatically in the position shown in FIG. 4, freeing both hands of the operator for rescue operations.

A further feature of the invention resides in the fact that when the operator displaces the brake means 70 from its braking position of FIG. 4 toward its release position of FIGS. 2 and 3, it is not essential that the operator turn the brake means all the way to its release position. As the brake means 70 approaches the braking position thereof shown in FIG. 4 there will be greater resistance to movement downwardly along the rope with consequent slowing down of the rate of downward movement, whereas as the braking means 70 approaches the release position of FIGS. 2 and 3 the speed of downward movement of the load will increase, so that by selectively positioning

the brake means between its braked position of FIG. 4 and its released position of FIGS. 2 and 3 it is possible for the operator to control the speed of movement downwardly along the rope.

FIGS. 9-11 show another embodiment of a structure according to the invention which corresponds generally to that of FIGS. 1-4. The embodiment of FIGS. 9-11, however, is particularly suitable for use with relatively heavy ropes, whereas the embodiment of FIGS. 2-4 can be used with lighter ropes. The friction slider means 90 of FIGS. 9-11 is substantially identical with the friction slider means 40. It differs therefrom in that the friction slider body 92 is provided at its thinner right side, as viewed in FIG. 9, which defines the opening 94 with an interruption accommodating a hinged gate 96 through which the rope 100, which is heavier than the rope 22, can pass into the opening 94. For this purpose the body 92 pivotally carries the gate 96 which has an inclined end 98 preventing it from moving to the right beyond the position shown in FIG. 9. At its lower pivoted end the gate 96 is formed with a detent recess receiving the spring-pressed free end of a detent element 102 which is situated in a suitable bore of the member 92, as indicated in FIG. 9, so that the gate 96 will be releasably maintained in a position shown in FIG. 9. However the operator can at any time turn this gate inwardly toward the opening 94 so as to introduce or remove a rope 100 therefrom, and in this way the rope can have any desired number of turns passing around the wider portion 104 of the body 92.

Except for these differences the friction slider means 90 of FIG. 9 is identical with that of FIGS. 2-4.

The braking means 110 of FIGS. 9-11 is also substantially identical with the brake means 70. The only differences reside in the fact that a spring means 112 is provided for urging the brake means 110 to its braking position, and an additional friction element 114 is provided to frictionally engage the rope, as indicated at the lower part of FIGS. 9 and 10.

The spring means 112 includes a pair of wire springs 116 each fixed at one end 118 to opposed side faces of the body 120 which constitutes the brake means 110. The wire springs 116 are also fixed to the ends of the pivot pin 122 which extends through the bore of the bottom of the body 92 which is received between the upper ears of the body 120, these features being the same as those described above in connection with FIGS. 2-4 at the interconnection between the friction slider means and the brake means. The springs 116 are under stress in the position of the parts shown in FIG. 10 and seek to turn the body 120 toward its braking position shown in FIG. 11, so that with this construction the springs 112 urges the brake means 110 toward its braking position, and full reliance need not be made on the pulling upwardly of the brake means by the rope 100 itself. Therefore this construction is particularly suitable for use with a relatively heavy rope.

In order to increase the frictional contact with such a relatively heavy rope, the lower free, bored end 126 of the body 120 carries in its bore an elongated, bowed leaf spring 128 having its free ends curved backwardly upon itself and situated in suitable notches of the body 120, as indicated most clearly in FIGS. 9 and 10, so that in this way the central bowed portion of the leaf spring 128 will press against the rope 100 to increase the frictional contact therewith, and this also will contribute to prevention of continued sliding movement of the brake means downwardly along the rope when the brake means is released by the operator. Therefore, the increased friction achieved by the leaf spring 128 will also contribute to automatic movement of the brake means 110 into the braking position thereof shown in FIG. 11 when the braking means is released by the operator.

Otherwise the structure of FIGS. 9-11 is identical with that of FIGS. 2-4 and therefore is not further described.

A further embodiment of the invention is illustrated in

FIGS. 5-8. This embodiment is shown in cooperation with a rope 200 which can be fixed to any selected support in the same way as the rope 22 described above.

The rope 200 is threaded through the friction slider body 202 in the manner shown most clearly in FIG. 5, this body 202 forming the friction slider means 204 of this embodiment. The body 202 is integral with an attaching means 206 in the form of a metal loop through which the harness 208 can pass in the manner shown most clearly in FIGS. 5 and 6, and the bands of the harness are bolted to the body 202 by way of the bolt and nut assembly 210.

The body 202 is formed with the rope-receiving passages extending inwardly toward the face 212 of the body 202 (FIG. 6) from the face 214 thereof. These rope-receiving recesses, however, terminate short of the face 212, and at its face 214 the body 202 removably carries a cover 216 which can be releasably fastened by way of screws 218 received in threaded bores 220 formed in the body 202. Thus, the plate 216 will retain the rope in the rope-receiving recesses of the body 202.

The rope-receiving recesses of the body 202 include a curved entrance recess 222 where the body turnably carries a guide roller 224 to facilitate the entrance of the rope 200 into the entrance recess 222. This curved recess 222 is situated at its end 226 opposite one end of a substantially U-shaped recess 228 which extends around a substantially drop-shaped guide member 230 around which the rope 200 passes. Between the end 226 of the recess 222 and the end 232 of the recess 228 there is a free space across which the rope 200 extends.

Beside the curved recess 222 the body 202 is formed with another substantially U-shaped recess 234 which is substantially identical with the curved recess 228 and also extends around a core or drop-shaped guide member 236. Thus, from the recess 228 the rope will pass across a free space formed between portions 240 and 242 of the body 202 into the curved recess 234 and around the guide body 236 with respect to which the rope can slide. After again passing downwardly across the free space 244 situated between the upper portion 240 and the lower portion 242 of the body 202, as viewed in FIG. 5, the rope 200 extends out through an elongated bore 246 which is formed in the body 202.

Between its body portions 240 and 242, the body 202 is provided with a portion 248 of substantially L-shaped profile, as shown in FIG. 8, and it is this portion 248 which defines with the portions 240 and 242 the free space 244 through which the three passes of the rope 200 extend in the manner described above and shown in FIG. 5.

The portion 248 of L-shaped profile terminates in an upper edge 250 and a lower edge 252 (FIG. 5), and these edges define with the body portions 240 and 242, respectively, spaces which receive upper and lower limbs 254 and 256 of an endless body 258 of substantially rectangular configuration having the vertically extending limbs 260 and 262 extending between the horizontal limbs 256 which are received in the spaces defined on the one hand between the body portion 240 and the upper edge 250 of the body portion 248 and on the other hand between the body portion 242 and the lower edge 252 of the body portion 248.

As is indicated in FIGS. 6 and 8, the limbs 256 of the rectangular endless body 258 are respectively formed with elongated slots 264 through which the passes 268 of the rope 200 freely extend in the manner shown most clearly in FIG. 6.

The limb 262 of the body 258 is pressed against by three compression springs 270 which are respectively accommodated in recesses 272 formed in the body 202 between its upper portion 240 and its lower portion 242, so that in this way the springs 270 urge the body 258 to the left from the position of FIG. 5 to the position of FIG. 7. This body 258 forms the brake means of the embodiment of FIGS. 5-8, and when the operator releases the body 258 the springs 270 will expand to the position

shown in FIG. 7 where the limb 262 presses the passes 268 against part of the portion 248 of the body 202, as indicated in FIG. 7, so as to achieve the required braking of the movement downwardly along the rope 200. The operator can, however, pass his fingers completely around the device just below the harness attachment means 206 so that with the palm of his hand the operator can press on the limb 260 to displace the brake means 258 to the right, as viewed in FIG. 7, in opposition to the springs 270 from the position of FIG. 7 into the position of FIG. 5 which is the release position, and now the load will again move downwardly around the rope. In this case also by selectively locating the brake means between the fully braked position of FIG. 7 and the fully released position of FIG. 5 it is possible to control the speed of movement downwardly along the rope.

In the example illustrated in FIGS. 5-8 there are three passes 268 of the rope 200 extending through the brake means to be pressed against each other by the brake means for the purpose of achieving the braking action in the manner shown in FIG. 7. However, it is clear that in accordance with the invention there may be any desired number of passes of the rope which are pressed against each other by the brake means of the embodiment of FIGS. 5-8. Furthermore, it is possible to provide, instead of fixed rope guides 236 and 230, guides which include rollers which reduce the friction and which may be used with heavier ropes than the rope 200. Such a roller could also be located at the upper end of the discharge bore 246 in order to guide a rope through the latter.

It will thus be seen that with all embodiments of the present invention when the operator releases the structure it will automatically brake itself so that it will stop its movement downwardly along a rope, and thus with the structure of the invention it is possible for the operator to terminate his downward movement at any desired elevation while leaving both hands free to carry out any desired operations at the elevation at which the operator releases the structure.

What is claimed is:

1. For use in a load-lowering assembly, friction slider means for frictionally engaging and sliding down a rope and for carrying a load which urges said slider means downwardly along the rope, and manually operable brake means operatively connected to said slider means for automatically assuming a braking position braking movement of said slider means and a load carried thereby downwardly along the rope, said brake means being manually displaceable with respect to said slider means from said braking position to a release position releasing said slider means for movement downwardly along the rope, said slider means and brake means respectively including a pair of elongated bodies both of which are formed with openings through which the rope passes for frictionally engaging said bodies, said slider means having an upper end and said brake means being pivotally connected to said slider means below the upper end of said slider means, and said brake means when in said release position thereof extending downwardly from the pivotal connection of said slider means and brake means to each other, said brake means when released by the operator coacting with the rope for responding automatically to tension in the rope to be pulled upwardly by the tension in the rope to said braking position, said brake means when in said braking position thereof extending upwardly from said pivotal connection and being situated alongside of said friction slider means.

2. The combination of claim 1 and wherein said bodies respectively have projections directed toward each other when said brake means is in said braking position thereof, and said projections engaging and pressing against the

rope to increase the frictional pressure between the latter and said bodies when said brake means is in said braking position.

3. The combination of claim 2 and wherein said projections are formed with openings through which the rope passes.

4. The combination of claim 3 and wherein said projection of said body of said brake means is situated beneath said projection of said body of said friction slider means when said brake means is in said braking position thereof.

5. The combination of claim 4 and wherein said body of said brake means is formed with a shoulder which becomes situated over said projection of said body of said slider means when said brake means is in said braking position thereof, said shoulder engaging the rope and deflecting the latter over said projection of said body of said slider means when said brake means is in said braking position.

6. The combination of claim 1 and wherein a spring means urges said body of said brake means to turn upwardly to the braking position.

7. The combination of claim 1 and wherein said body of said brake means is formed with an opening through which the rope passes and said body of said brake means carrying in said latter opening thereof a friction element against which the rope frictionally slides.

8. The combination of claim 1 and wherein the rope is coiled about said body of said friction slider means at one side of said opening thereof, said body of said friction slider means being formed at the other side of said latter opening with an interruption through which the rope may be inserted into the latter opening.

9. For use in a load-lowering assembly, friction slider means for frictionally engaging and sliding down a rope and for carrying a load which urges said slider means downwardly along the rope, and manually operable brake means operatively connected to said slider means for automatically assuming a braking position braking movement of said slider means and a load carried thereby downwardly along the rope, said brake means being manually displaceable with respect to said slider means from said braking position to a release position releasing said slider means for movement downwardly along the rope, said friction slider means including a body formed with an opening through which the rope is adapted to pass with said rope coiled around said body at one side of said opening thereof, said body carrying at the opposite side of said opening a pivotally mounted gate giving access to said opening and a spring-pressed detent releasably maintaining said gate in a position closing said opening.

10. The combination of claim 9 and wherein said body is wider at said one side thereof than at said opposite side thereof.

11. The combination of claim 9 and wherein said gate is turnable into said opening for giving access thereto and has an end overlapping and engaging said body to prevent turning of said gate outwardly beyond said opening.

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